

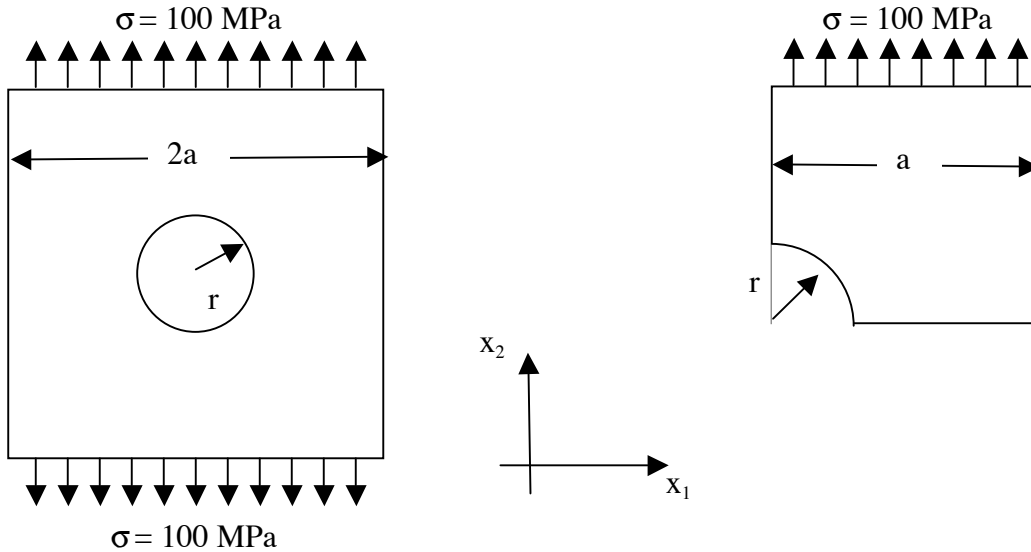
Computer Assignment

Due Friday, Nov. 16

1. Practice the use of ABAQUS/CAE by following the document *1-3 CAE Example.pdf* which is available on course website and iMechanica.

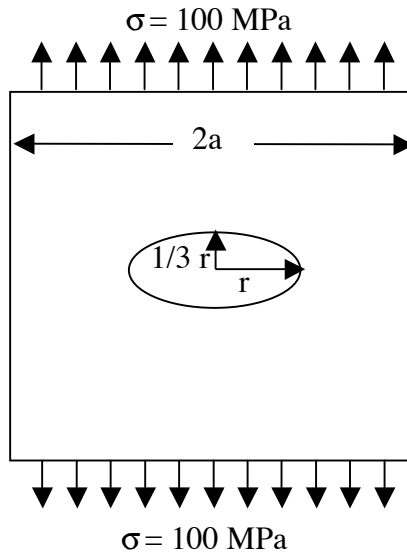
Devise 2 (or more) loaded structures of your own and create models using ABAQUS/CAE. Make your first structure simple and statically determinate, so that you can easily verify any answers from the computer analysis of its deformations and stresses. Make the second one(s) interesting.

2. Stressing a plate with a circular or elliptical hole



A square plate with sides length $2a$ and a circular hole with radius r is shown above. The plate is under remote tensile loading, σ , as shown. A finite element model of the figure on the figure on the right, which takes advantage of the symmetry, is given in the input file *strnrhl-elps.inp* (see the end of this assignment for input file).

- (1) Run the input file *strnrhl-elps.inp*. Notice that there is no loading given in the static step. This input file just creates the mesh for the figure shown on the right. Notice that an element set called “LOADED” is defined. Make sure you understand the commands used to generate the mesh (*NGEN, *NFILL, *ELGEN)
- (2) Modify the input file used in part (1) to analyze the problem of a plate under remote tensile loading.
 - a. Use the element set “LOADED” to apply the appropriate loading on the boundary. One way to do this is to define a distributed load on the element faces at the boundary using the *DLOAD command.
 - b. Modify the step definition so that a perturbation step is used.
 - c. Print and Run your modified (and well commented) input file. (NOTE: in *strnrhl-elps.inp* the output requests have been commented out – you will have to change this to get output to the data and output database files)
 - d. Use ABAQUS/Viewer to examine and prepare plots of your solutions, these should include: the deformed shape, contour plots of stresses, contour plots of strains.
- (3) The input file *strnrhl-elps.inp* uses 8-noded isoparametric elements. Write an input file to solve the same problem using constant-strain-triangular elements. Run the file and examine similar output as mentioned in part (2).



- (4) The geometry for parts (1)-(3) involved a circular hole in a square plate. Now consider a plate with an elliptical hole. (See the attachment for theoretical analysis of an elliptical hole.) Modify the input files used in parts (2) and (3) so that the round hole becomes an elliptical hole, with dimensions shown above. To do this, you can use the command

*NMAP:

*NMAP, NSET = MAPSET, TYPE=RECTANGULAR

0,0,0,1,0,0

0,1,0

0,0.333,0

Look up this command so that you understand how it changes the nodal coordinates. Repeat parts (2) & (3) for this configuration.

- (5) The solution for an infinite plate with a circular hole under remote tensile stress loading σ is derived in Rice, *Solid Mechanics*, pp. 65-68, and some results from the solution for an elliptical hole are given there too. Provide a *clear, careful* and *detailed* comparison of your numerical solutions from (2), (3), and (4) to those results. E.g., compare to predicted maximum stresses, to σ_θ versus r for the circular case and to displacements of the hole boundary along the rays $\theta = 0^\circ$ and 90° . How could you determine whether differences were due primarily to inaccuracies of the numerical procedure, or to the infinite versus finite overall plate dimensions?

strnhl-elps.inp

```

*****
**
** written by James R. Rice
**
** last modified: 10/26/2006
**          by: Nanshu Lu
**
*****
** brief description:
**
** Units used in what follows are
** millimeters (mm) for lengths,
** Newtons (N) for forces, and hence
** N/mm^2 = MPa (Pascals) for moduli and
** stresses
**
*****
** heading information
**
*****
*HEADING
stressfield in sq elastic plate with hole
**
*****
** Parameter Definitions
**
*****
*PARAMETER
E = 2E5
nu = 0.30
**
** E: Young's Modulus (MPa)
** nu: Poisson ratio
**
*****
** Node definitions
**
*****
*NODE
1, 1., 0.
9,.707, .707
17, 0., 1.
441,4., 0.
449,4., 4.
457, 0., 4.
1000, 0., 0.
*NGEN, LINE=C, NSET=HOLE1
1, 9, 1, 1000
*NGEN, LINE=C, NSET=HOLE2
9, 17, 1, 1000
*NSET, NSET=MAPSET
HOLE1, HOLE2
*NGEN, NSET=EDGE1
441, 449, 1
*NGEN, NSET=EDGE2
449, 457, 1
*NFILL, BIAS=0.88, TWO STEP, NSET=NALL1
HOLE1, EDGE1, 22, 20
*NFILL, BIAS=0.88, TWO STEP, NSET=NALL2
HOLE2, EDGE2, 22, 20
*NSET, NSET=HORSYM, GENERATE
1, 441, 20
*NSET, NSET=VERSYM, GENERATE
17, 457, 20
**
*****
** Element Definitions
**
*****
**
** ELEMENT, TYPE=CPS8
1, 1, 41, 43, 3, 21, 42, 23, 2
**
** ELGEN, ELSET=ELALL
1, 8, 2, 1, 11, 40, 10
**
** ELSET, ELSET=LOADED, GENERATE
105, 108, 1
**
*****
**
** Material Definitions
**
*****
**
** SOLIDSECTION, ELSET=ELALL, MATERIAL=ELAS
** MATERIAL, NAME=ELAS
** ELASTIC
<E>, <nu>
**
*****
** Boundary Conditions
** look up the below boundary conditions
** so that you understand them, and add
** comments.
**
**
*****
**
** BOUNDARY
VERSYM, XSYMM
**
**
** BOUNDARY
HORSYM, YSYMM
**
*****
**
** Step 1: apply loading

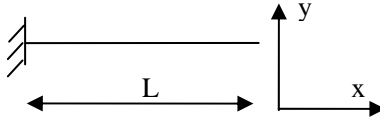
```

```

**
*****
**
*STEP, NAME=STEP-1
*STATIC
**
*****
**
** Loads
**
*****
**
**
**
*****
**
** OUTPUT REQUESTS
**
*****
**
** The following option is used to write
** output to the output database file
** (for plotting in ABAQUS/Viewer)
**
***OUTPUT, FIELD, VARIABLE=PRESELECT
**
** The following options are used to
** provide tabular printed output of
** element and nodal variables to
** the data and results files
**
***EL PRINT, POSITION=CENTROIDAL
** S,COORD
**
**
***NODE PRINT
** U,COORD
**
*****
**
**
*END STEP

```

3. When you use ABAQUS CAE to solve a natural frequency problem, the way to do this kind of analysis is: When you create a step: Select procedure type Linear perturbation → Select Frequency. All other things are similar to what you have done for static problems.



Use ABAQUS CAE to get the natural frequency for a rod under **axial** loading. Show that the

analytical solution is given by $f_1 = \frac{1}{4L} \sqrt{\frac{E}{\rho}}$. Use 1 element, 2 elements... till 10 elements to get

the first natural frequency. You can model this rod as a beam. Hint: to avoid getting the bending mode, displacement in the y direction along the beam should be fixed so that the beam can only

deform in the x direction. Plot your dimensionless frequency $\frac{f_1 L \sqrt{\rho}}{\sqrt{E}}$ as a function of the

element number.

σ^∞ in the 2 direction as above. This also describes the tunnel cavity of elliptical cross section under plane strain. Let a denote the semi-axis of the ellipse along the 1 direction and b denote that along the 2 direction; the equation of the ellipse is $X_1^2/a^2 + X_2^2/b^2 = 1$. It is then found that the concentration of stress at points of the hole boundary intersected by the X_1 axis is $\sigma_{22} = (1 + 2b/a)\sigma^\infty$, which can be rewritten as $\sigma_{22} = (1 + 2\sqrt{a/\rho_{tip}})\sigma^\infty$. In the latter form, $\rho_{tip} = b^2/a$ is the radius of curvature of the hole boundary, at the tip of the hole at $X_1 = \pm a$. This illustrates a result of general validity for notches with relatively small root radii compared to length: The elevation of stress over the value (σ^∞) in absence of the notch is, very approximately, given by $2\sqrt{a/\rho_{tip}}\sigma^\infty$ in all cases, where a is the half length of an internal notch like the elliptical hole just discussed, and is the length of a notch that has been cut in from the free surface of a solid. Thus good engineering design is always sensitive to the stress concentrating effect of holes and, especially, notches or other cut-outs of small root radius, avoiding them where possible. This is a lesson reinforced by the bitter experience of many structural failures beginning at unrecognized locations of stress concentration. In the particular case of the elliptical hole, the stress induced along the hole boundary where it is intersected by the X_2 axis is $\sigma_{11} = -\sigma^\infty$, independent of the b/a ratio.

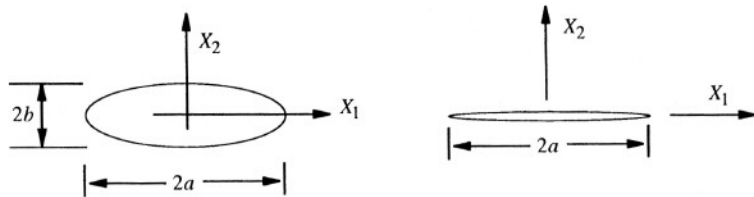


Figure 15. Elliptical tunnel hole, and limit as a flat crack.

Inclusions. Points around the boundary of the elliptical hole just discussed are found to displace according to the equations

$$u_1 = -(\sigma^\infty / E')X_1, \quad \text{and} \quad u_2 = (1 + 2a/b)(\sigma^\infty / E')X_2$$

where X_1 and X_2 are coordinates of points on the boundary, and satisfy $X_1^2/a^2 + X_2^2/b^2 = 1$. Also, $E' = E$ for the plane stress model but $E' = E/(1 - \nu^2)$ for plane strain. These results show that if we had considered not a solid with an elliptical hole but rather a solid with a uniform elliptical

inclusion of another material, in this case a material with vanishing small elastic modulus compared to that of the surrounding solid (so that the situation in the surrounding solid is indistinguishable from that for the case of a hole), then that inclusion would undergo uniform strain, the strains ϵ_{ij}^{incl} within it being given by rewriting the above equations as $u_1 = \epsilon_{11}^{incl}X_1$ and $u_2 = \epsilon_{22}^{incl}X_2$ and noting that $\epsilon_{12}^{incl} = 0$. A little reflection on this result will convince one that if an inclusion of arbitrary but uniform and isotropic material properties (the stress-strain relation for the inclusion material need not even be linear) were placed in the hole, then the inclusion would undergo a uniform stress and strain that could be calculated from its material properties and the information given so far here.

This discussion generalizes to an important three-dimensional result discovered by J. D. Eshelby and which is this: Let a uniform, possibly anisotropic, linear elastic solid of infinite extent be loaded by a remotely uniform stress tensor and let it contain an ellipsoidal inclusion of a material of uniform but different mechanical properties. The inclusion material can even be such that, in its stress-free state, it takes an ellipsoidal shape which differs finitely from the stress-free shape of the ellipsoidal hole into which it is to be inserted. Eshelby's result is that, regardless of all these factors, the inclusion undergoes a spatially uniform stress and strain state. To develop that result, Eshelby first solved the *transformation* problem of a misfitting linear elastic inclusion of identical properties as those of the surrounding material. That involves taking an ellipsoidal region of a uniform solid and, at least conceptually, transforming its stress-free state by a homogeneous infinitesimal strain, without changing its elastic properties. A uniform stress field within the transformed zone then suffices to deform it back to its original shape, and that stress field can be maintained *in-situ*, without disturbance of the region outside the ellipsoid, by application of a suitable layer of surface force. Since the actual transformation problem to be solved has no agent to supply that layer of force, the strains everywhere can be calculated as those due to removing such a layer (i.e., applying a force layer of opposite sign) within an elastically uniform full space. One calculates then that the stress and strain state induced within the inclusion is uniform, and the rest of what is needed for arbitrary inclusions can be developed from there.

Crack as limit of elliptical hole. Consider again the two-dimensional problem of the elliptical hole under remotely uniform tension, and let the semi-axis b go to zero (Figure 15, at right) so as to define a flat Griffith crack lying along the X_1 axis on $-a < X_1 < +a$. In this case the stress concentration at the hole becomes unbounded so that it can fairly be objected that the discussion lies outside the proper realm of linear elasticity. However, as will be seen, it proves quite useful to continue with the linear elastic model and to learn about its stress singularities. Letting $\Delta u_2 = u_2^+ - u_2^-$ denote the crack opening gap (superscripts + and - denote the upper and lower